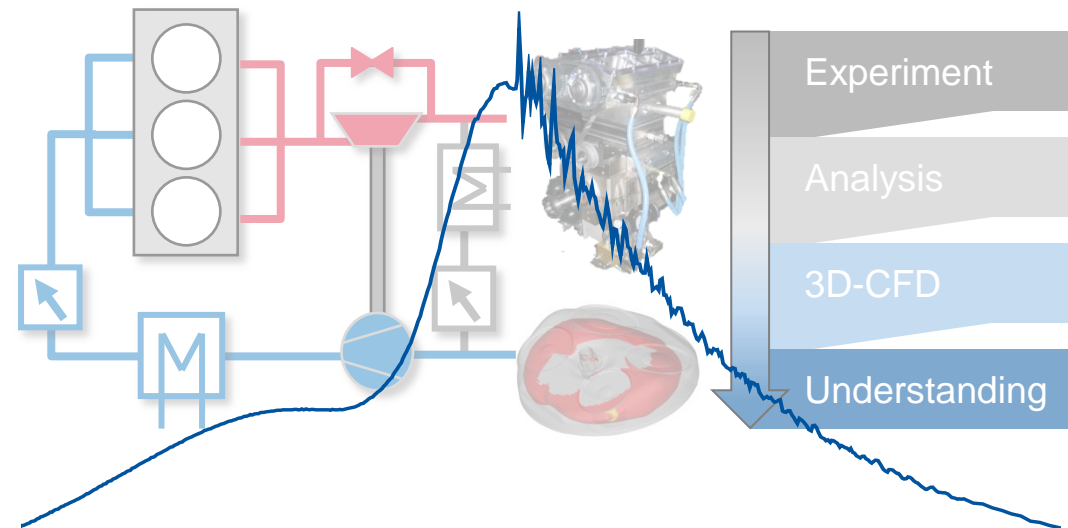




Institute for Combustion Engines  
RWTH Aachen University

Prof. Dr.-Ing. Stefan Pischinger

## NUMERICAL STUDY OF KNOCK INHIBITION WITH COOLED EXHAUST GAS RECIRCULATION



Bologna, March 20<sup>th</sup>, 2018

Max Mally

Dr.-Ing. Marco Günther

Prof. Dr.-Ing. Stefan Pischinger

# Agenda/Content

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- Introduction
- Experimental investigation
- Numerical investigation
- Summary and outlook

# Agenda/Content

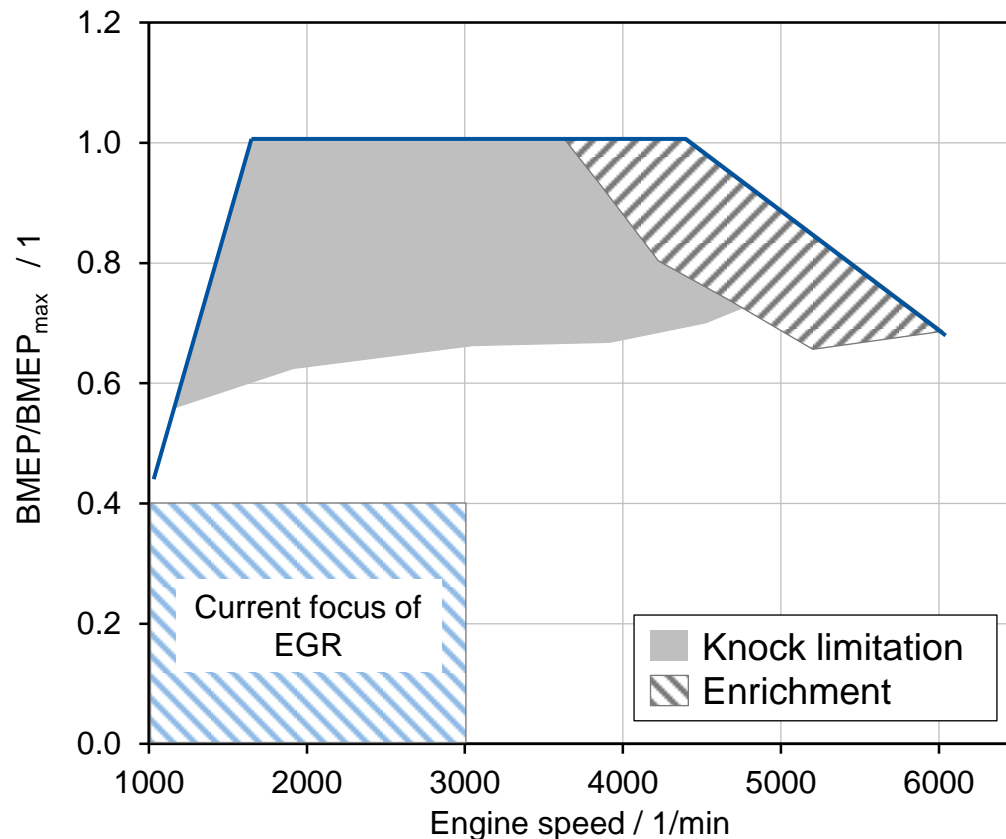
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# Introduction

## Simulation approach for layout of full load EGR combustion system needed

REDUCED EFFICIENCY OF CONVENTIONAL GASOLINE ENGINES DUE TO KNOCK AND FULL LOAD ENRICHMENT



EGR is a powerful measure to face these problems

- ⊙ However, development of a full load EGR combustion system is a complex task which requires reliable CAE
- ⊙ 3D-CFD simulation approach needed to predict combustion and knock under the influence of EGR
  - ✓ Experimental Database
  - ✓ Definition of simulation methodology
  - ✓ Validation of numerical models

# Agenda/Content

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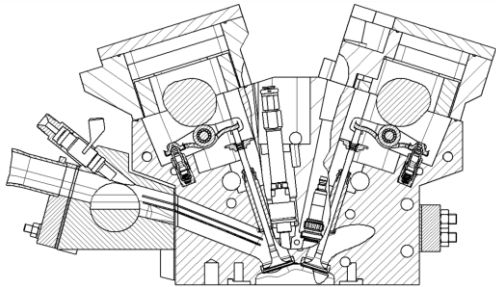
- Introduction
- Experimental investigation
- Numerical investigation
- Summary and outlook

# Experimental investigation

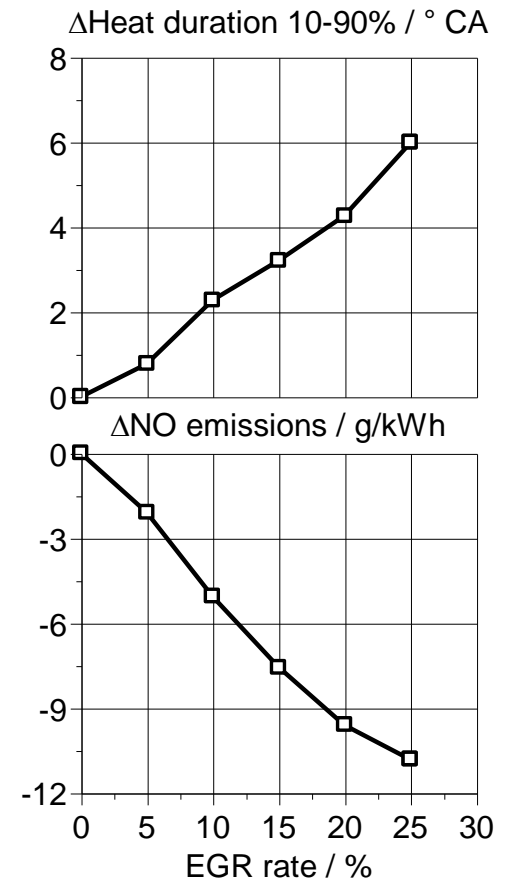
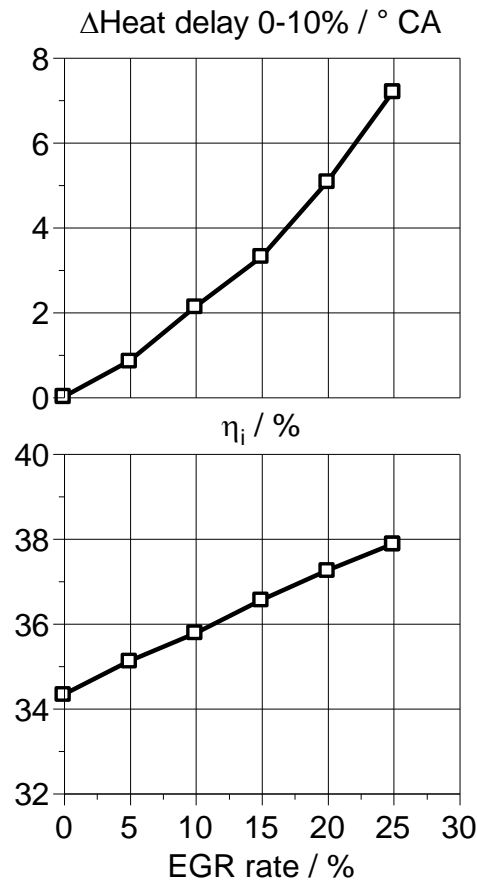
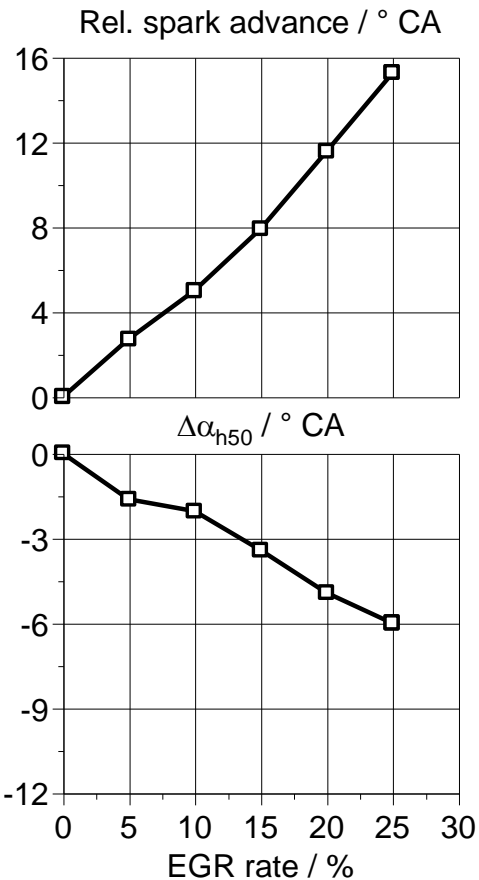
## Cooled EGR is a powerful measure to inhibit knock and increase efficiency

RON95E10       $p_{mi} = 16 \text{ bar}$        $\lambda = 1.0$   
 $p_{Rail} = 200 \text{ bar}$        $T_{Oil} = 90 \text{ }^\circ\text{C}$        $T_{Coolant} = 90 \text{ }^\circ\text{C}$

—□— 1500 1/min,  $T_{Intake} = 35 \text{ }^\circ\text{C}$



Engine specifications		
Stroke (s)	90.5	mm
Bore (D)	75	mm
s / D	1.207	1
Engine displacement	399	cm <sup>3</sup>
Connection rod length	152	mm
Piston-pin offset (thrust side)	0.5	mm
Compression ratio	10.9	1

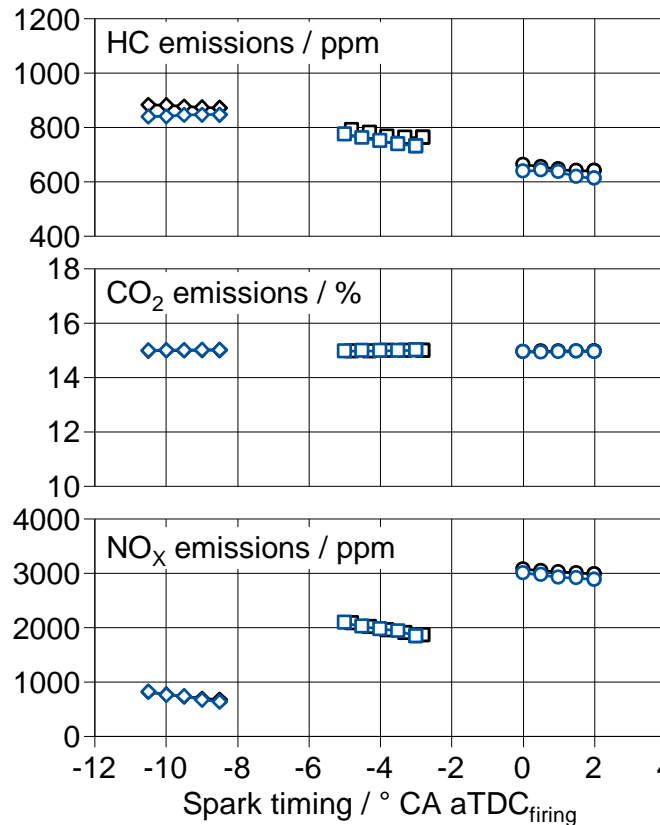
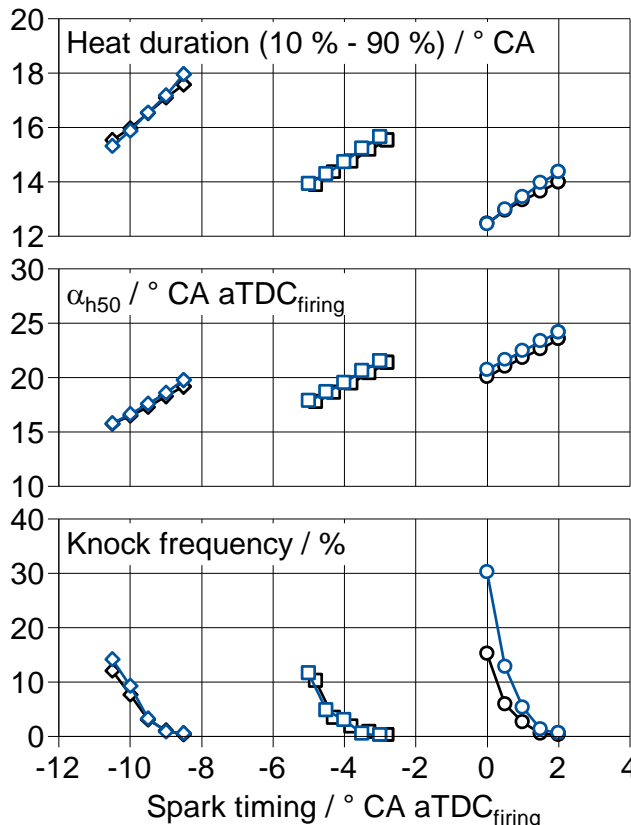


# Experimental Investigation

## Good agreement in terms of knocking and emissions with TRF+E surrogate

$n = 1500 \text{ 1/min}$     $p_{mi} = 16 \text{ bar}$     $\lambda = 1.0$   
 $p_{Rail} = 200 \text{ bar}$     $T_{Oil} = 90 \text{ }^\circ\text{C}$     $T_{Coolant} = 90 \text{ }^\circ\text{C}$   
 $T_{Intake} = 35 \text{ }^\circ\text{C}$

○ 0 % EGR   ○ RON95E10  
 □ 10 % EGR   □ TRF+Ethanol  
 ◇ 20 % EGR



	RON /-	MON /-	H/C /1
RON95E10	96.5	85.2	1.94
Isooctane	100	100	2.25
PRF95	95	95	2.25
TRF + Ethanol	95	86.6	1.93
	96.2	88.7	1.93

- ① 1) Properties according to blending rules
- ② 2) Measured properties from CFR engine
- ③ Validated kinetic mechanism available from ITV

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  - ⌵ Simulation setup
  - ⌵ Simulation results
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# Numerical investigation

## Numerical setup for simulation of mixture formation

### Mixture Formation

### Combustion

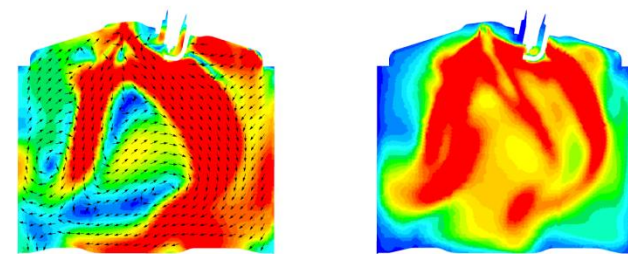
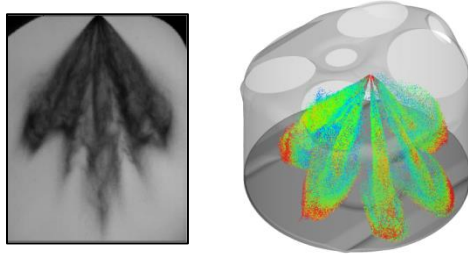
270° CA

660° CA

800° CA

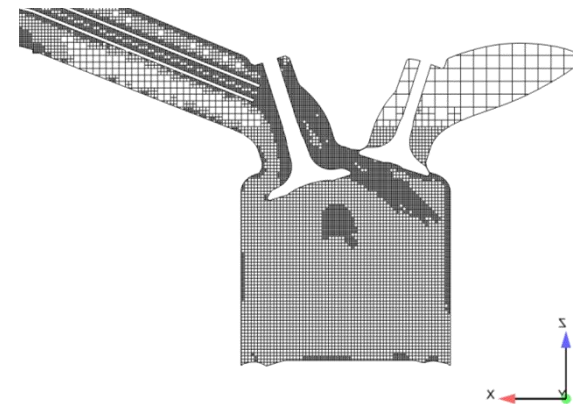
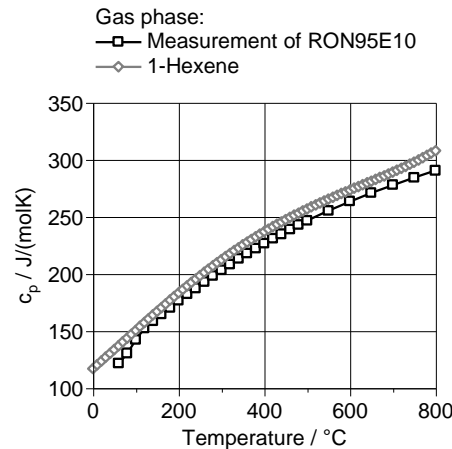
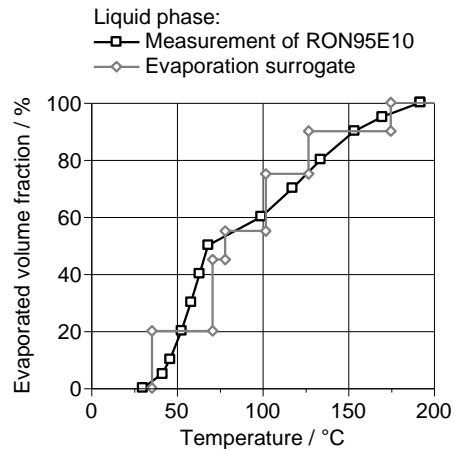
- Calibrated KH-RT model / Kuhnke Wall film model

- Standard k- $\epsilon$  turbulence model



- Dedicated surrogate for liquid phase

- Velocity AMR and Embedding (0.5 mm cells)



# Numerical investigation

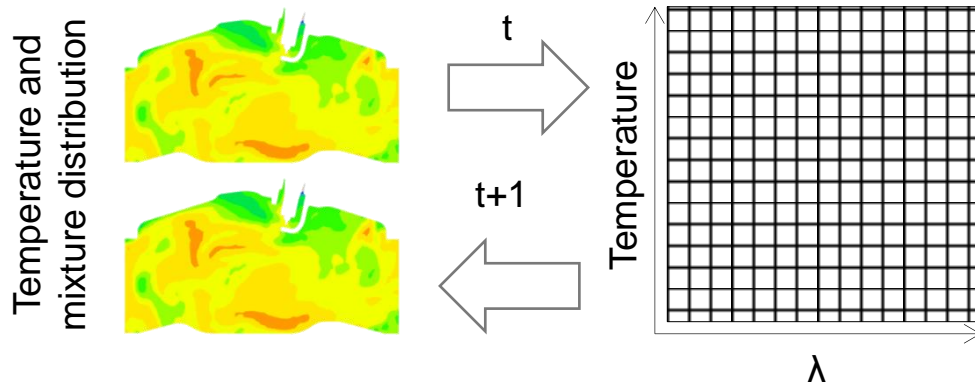
## Numerical setup for simulation of combustion and knock

Mixture Formation

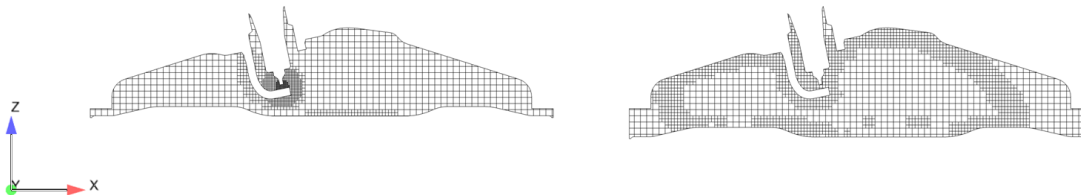
Combustion

270° CA

- ⊙ SAGE Chemistry Solver and Multi-Zone approach

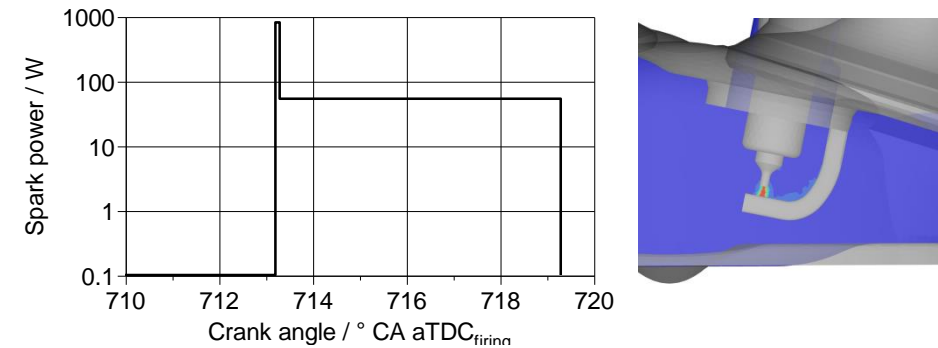


- ⊙ Spark embedding (0.0125 mm) and Temperature AMR (0.5 mm)

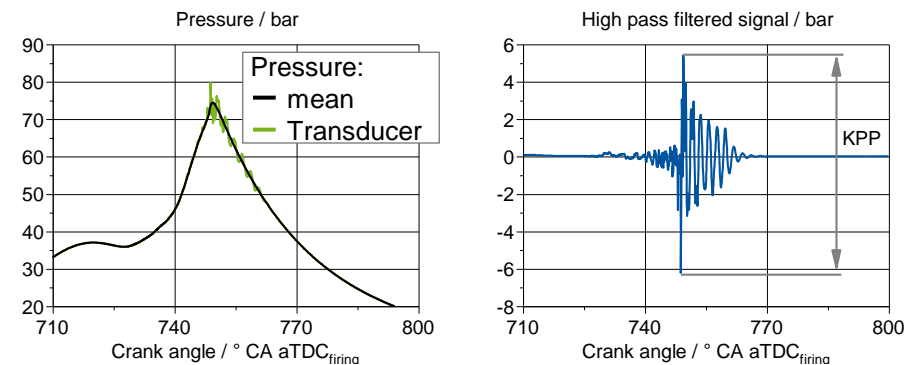


660° CA

- ⊙ Spark model (Energy line source)



- ⊙ Quantitative knock evaluation at transducer location



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# Numerical investigation

## Simulation results are feasible and confirm test bench knock limitation

SIMULATION RESULTS OF 2500 1/MIN, 0% EGR,  $T_{\text{INTAKE}} = 35 \text{ }^{\circ}\text{C}$

Simulation results:

$n = 2500 \text{ 1/min}$     $p_{\text{mi}} = 16 \text{ bar}$     $\lambda = 1.0$     $T_{\text{Intake}} = 35 \text{ }^{\circ}\text{C}$

$x_{\text{EGR}} = 0 \text{ } \%$

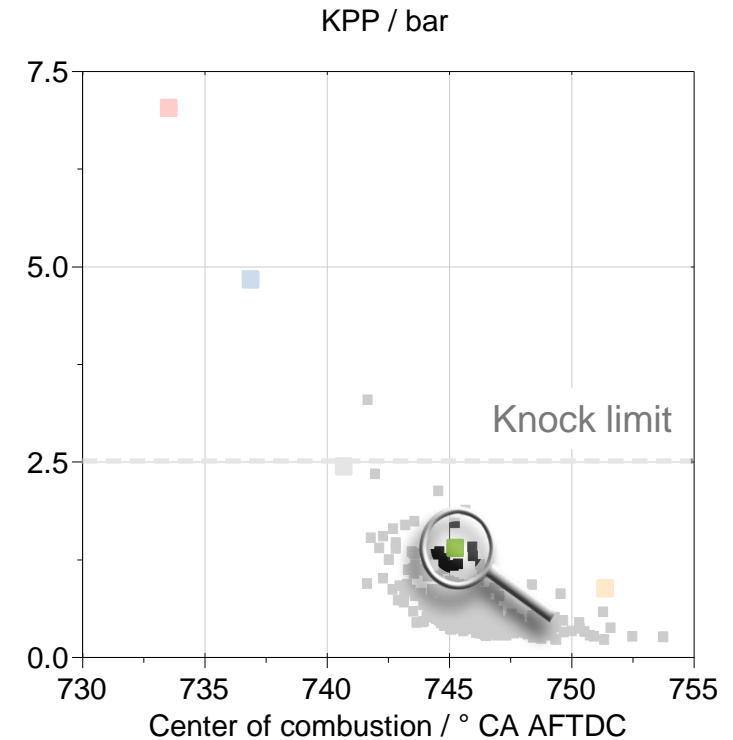
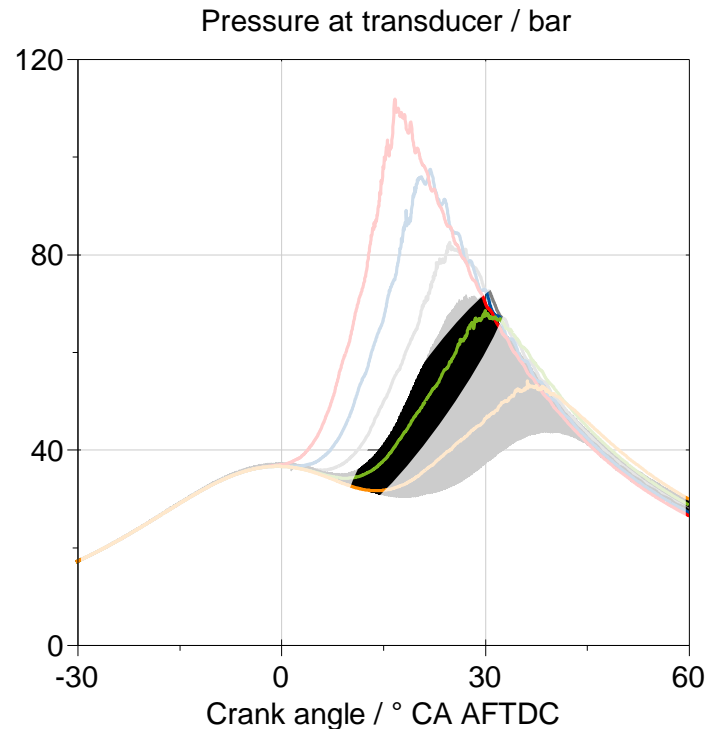
— ■ Experiment

Spark Timing

— #1 — #2 — #3

— #4 — #5

- ⊙ During testing cycle to cycle variations are observed
- ⊙ Fluctuations in combustion cannot be resolved with a RANS\* approach
- ⊙ Different combustion phasing in CFD is achieved via a virtual spark timing sweep
- ⊙ Combustion and knock limitation successfully reproduced with simulation approach

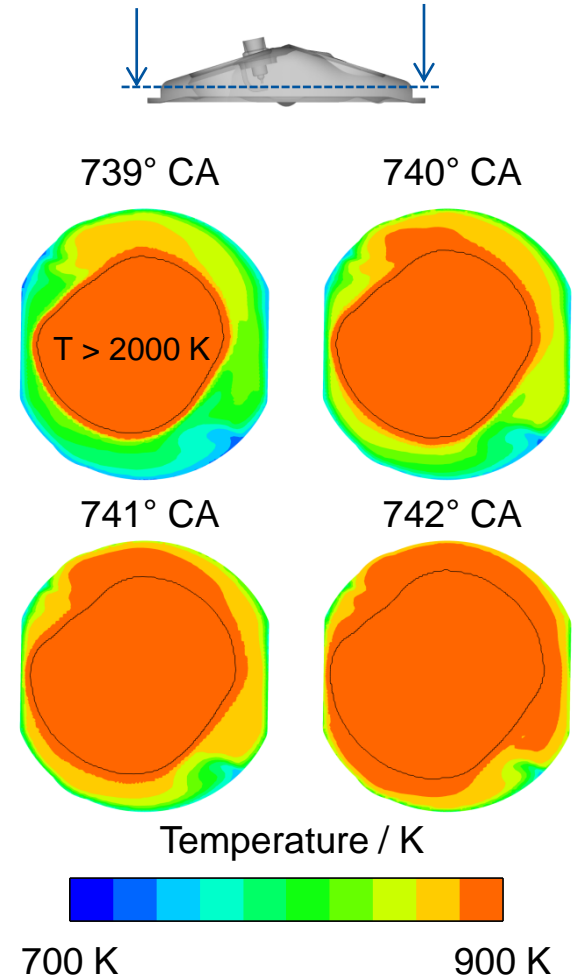
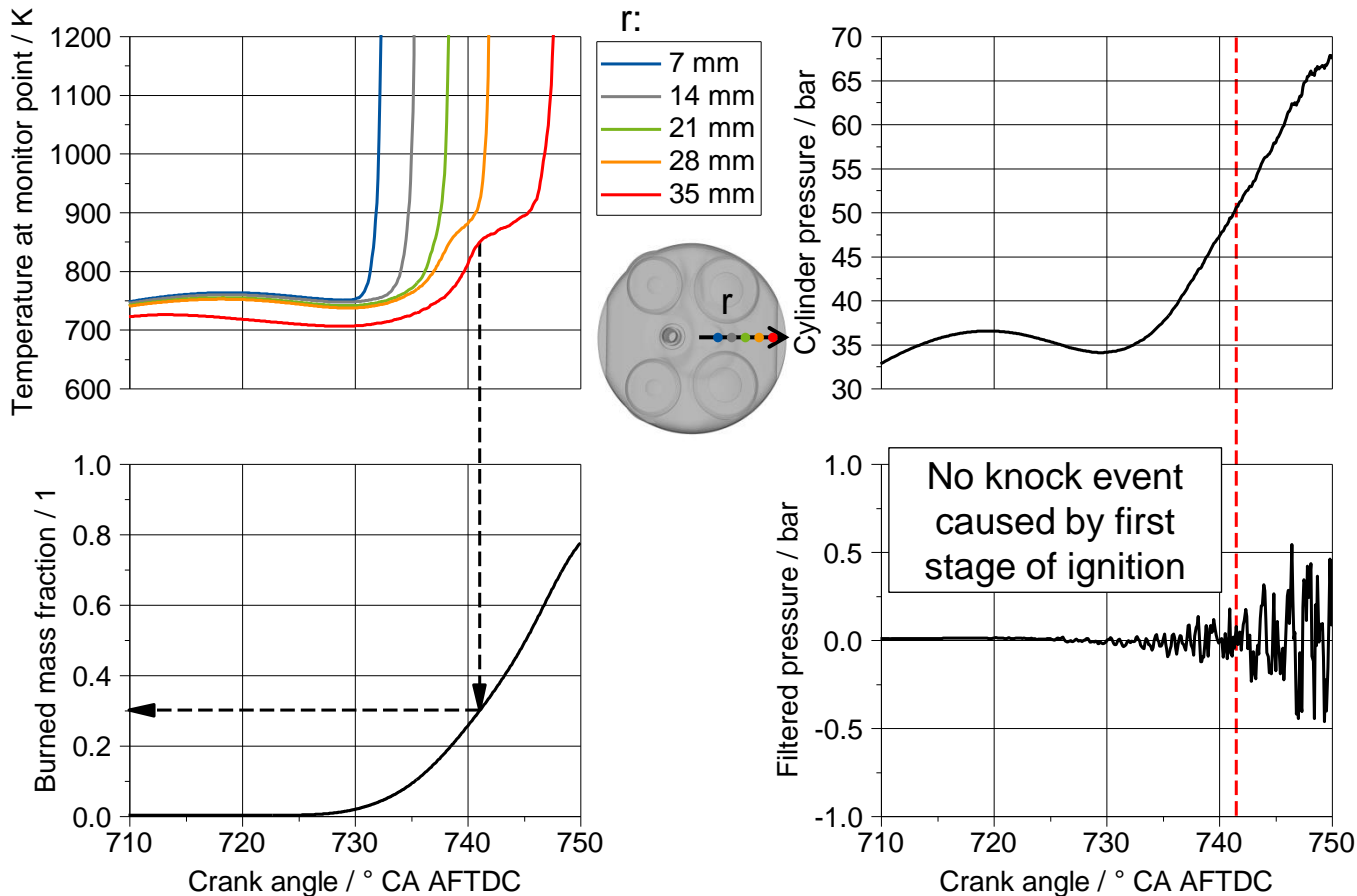


\*: Reynolds-Averaged Navier-Stokes

# Numerical investigation

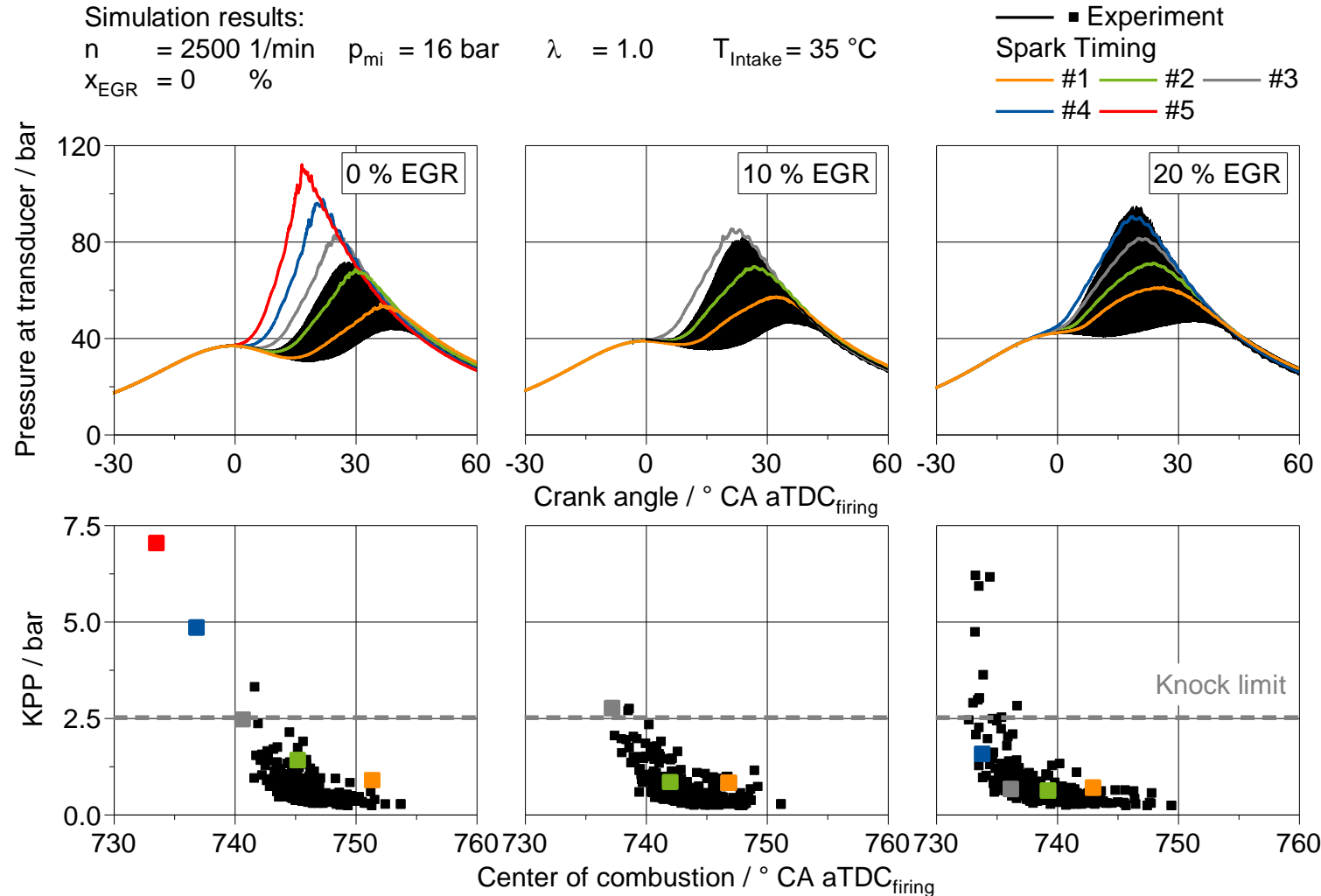
## First stage ignition observed early during combustion

SIMULATION RESULTS OF 2500 1/MIN, 0 % EGR,  $T_{\text{INTAKE}} = 35\text{ }^{\circ}\text{C}$



# Numerical investigation

## Effect of EGR on combustion is well captured by modelling approach

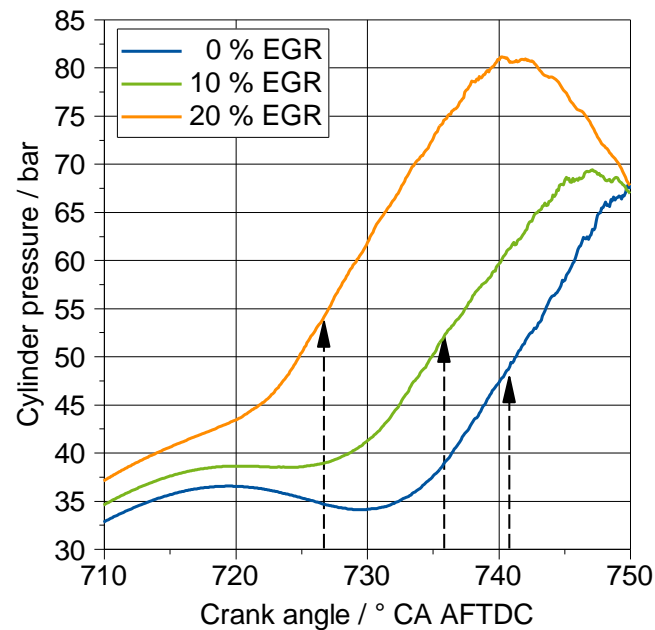
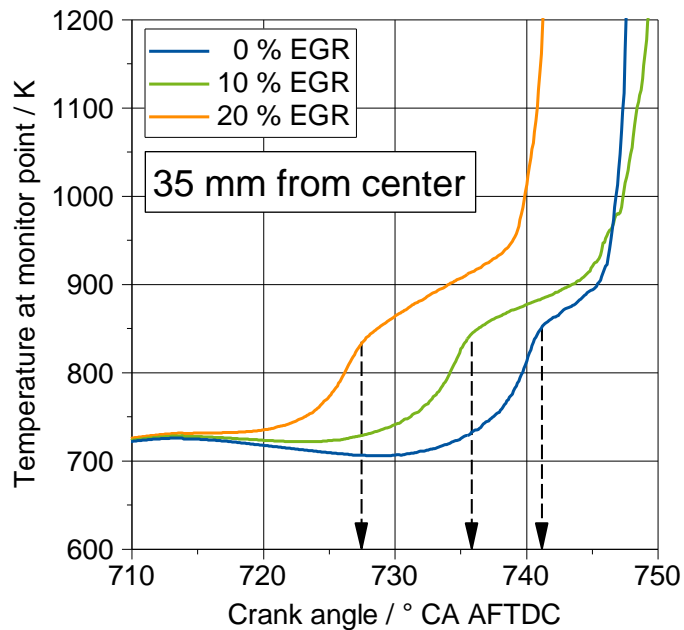


- ⊙ Knock limit well captured at 0 and 10 % EGR
- ⊙ At 20 % the knocking tendency is underestimated by the simulation model
- ⊙ The high cycle to cycle variation makes it difficult to capture the knock limit with a RANS approach

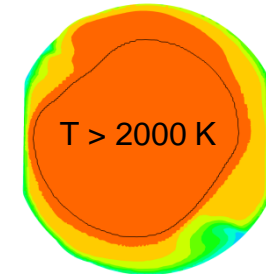
# Numerical investigation

## First stage ignition observed earlier when EGR is added

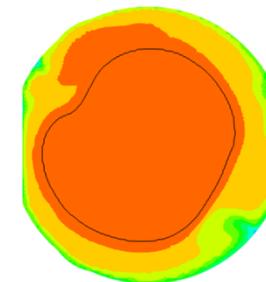
SIMULATION RESULTS OF 2500 1/MIN, 0-20 % EGR,  $T_{\text{INTAKE}} = 35^\circ\text{C}$



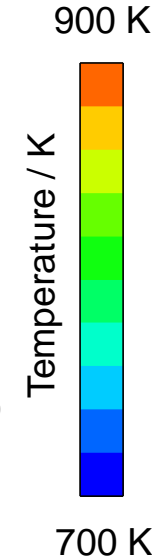
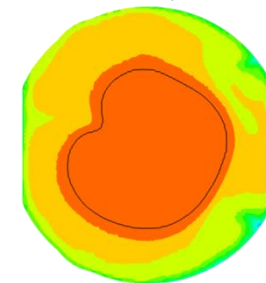
0 % EGR, 741° CA



10 % EGR, 736° CA



20 % EGR, 727° CA



- Despite similar temperature level the first stage of ignition is occurring earlier in the combustion simulation if EGR is added
- The cylinder pressure where first stage ignition is occurring is increasing with EGR

# Numerical investigation

## Simulation results are feasible and confirm test bench knock limitation

SIMULATION RESULTS OF 2500 1/MIN, 0% EGR,  $T_{\text{INTAKE}} = 35 \text{ }^{\circ}\text{C}$

Simulation results:

$n = 2500 \text{ 1/min}$     $p_{\text{mi}} = 16 \text{ bar}$     $\lambda = 1.0$     $T_{\text{Intake}} = 35 \text{ }^{\circ}\text{C}$

$x_{\text{EGR}} = 0 \text{ } \%$

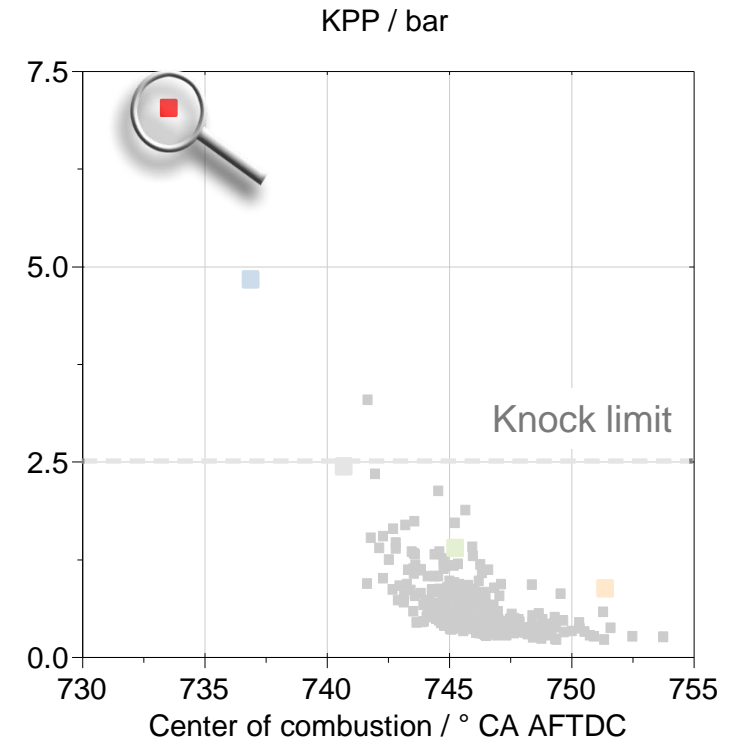
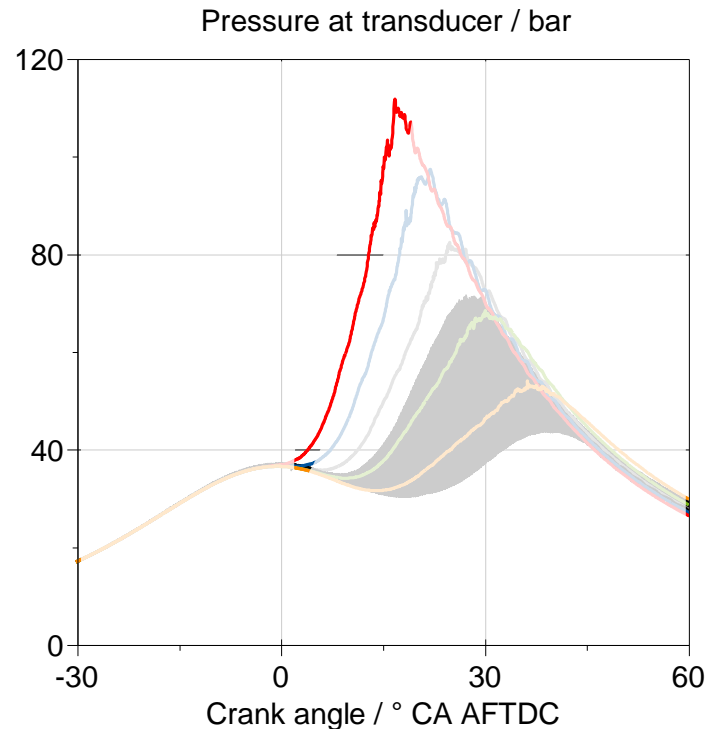
— ■ Experiment

Spark Timing

— #1 — #2 — #3

— #4 — #5

- ⊙ During testing cycle to cycle variations are observed
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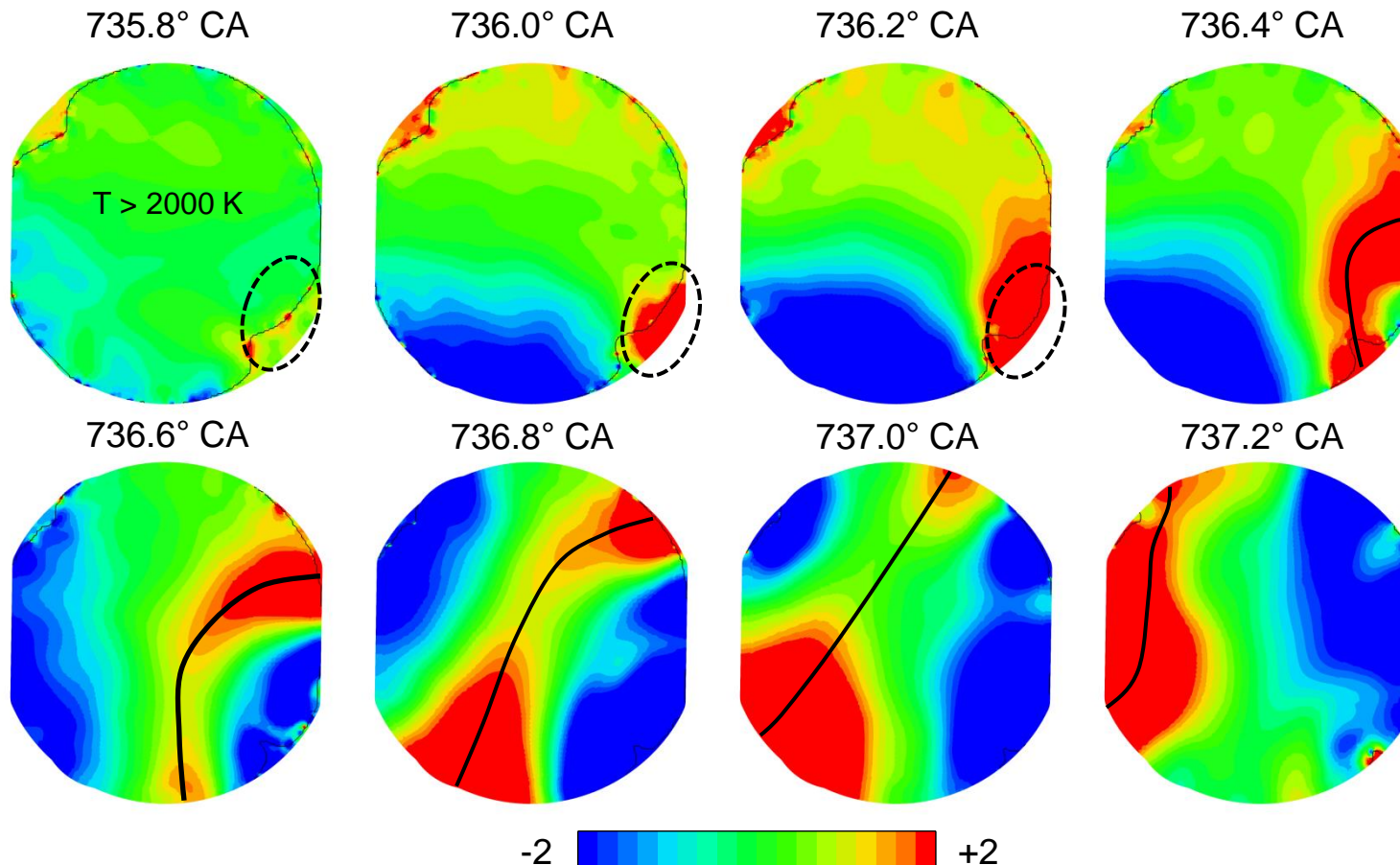
\*: Reynolds-Averaged Navier-Stokes



# Numerical investigation

## Severe knock caused by auto-ignition ahead of the flame front

PRESSURE WAVES ARE FORMED IN CASE OF FAST FLAME PROPAGATION (DETONATION)



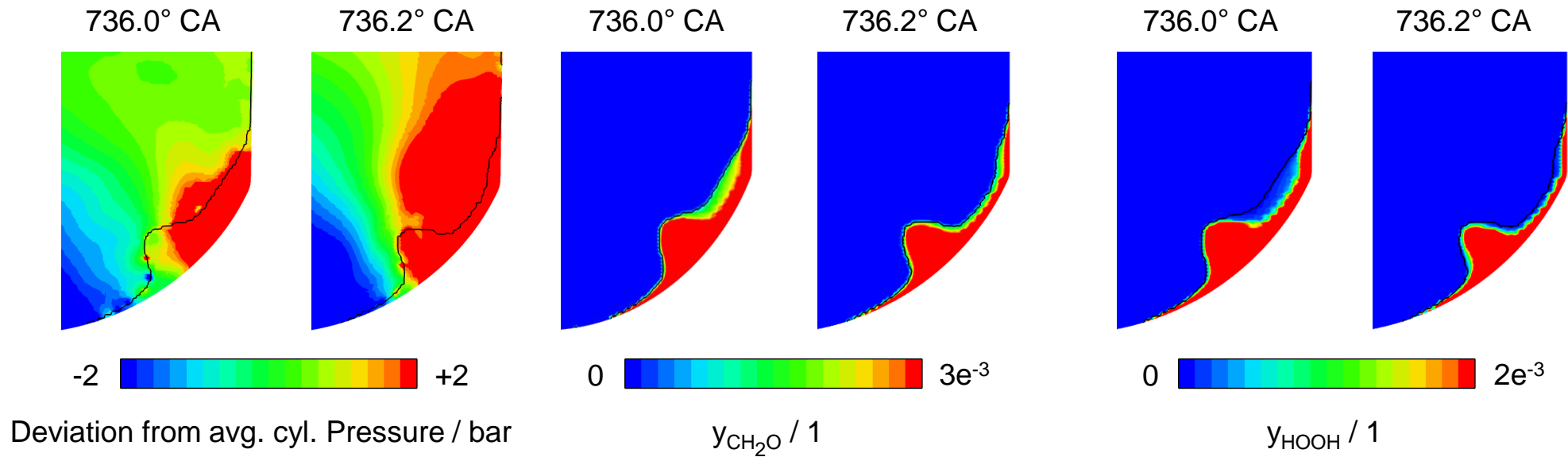
Deviation from avg. cyl. Pressure / bar

- In case of a severe knock event an extreme fast flame propagation is observed
- This is in-line with the detonation theory
- Due to the high heat release the local pressure cannot balance out with the surrounding environment
- The formed pressure waves travels through the combustion chamber and is reflected at the opposite wall

# Numerical investigation

## Formation of OH radicals by dissociation of HOOH leads to knock

AS  $\text{CH}_2\text{O}$  AND  $\text{HOOH}$  ARE CONSUMED AHEAD OF THE FLAME FRONT A PRESSURE WAVE IS FORMED

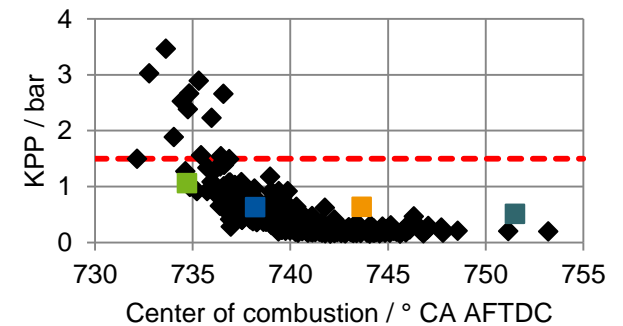
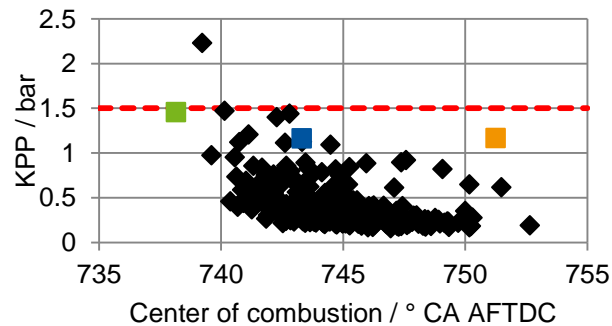
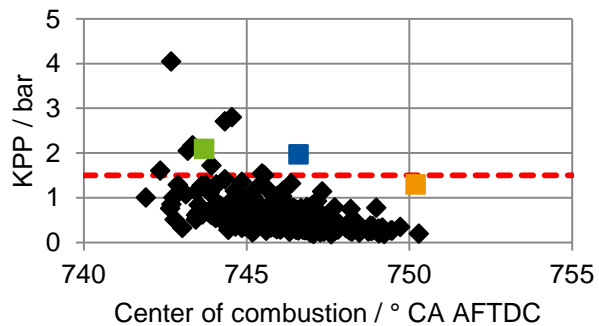
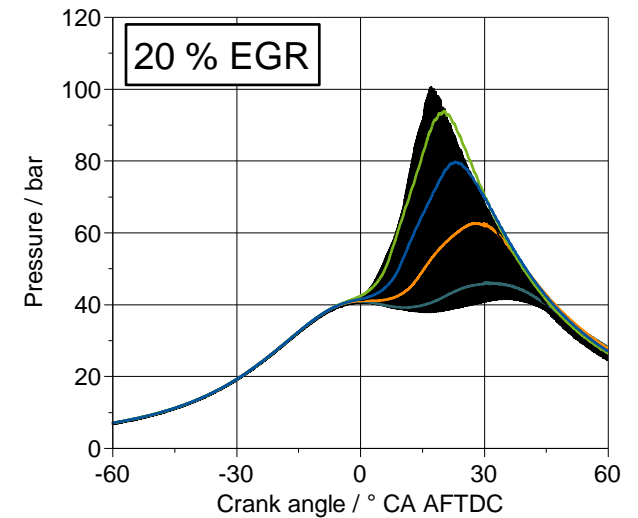
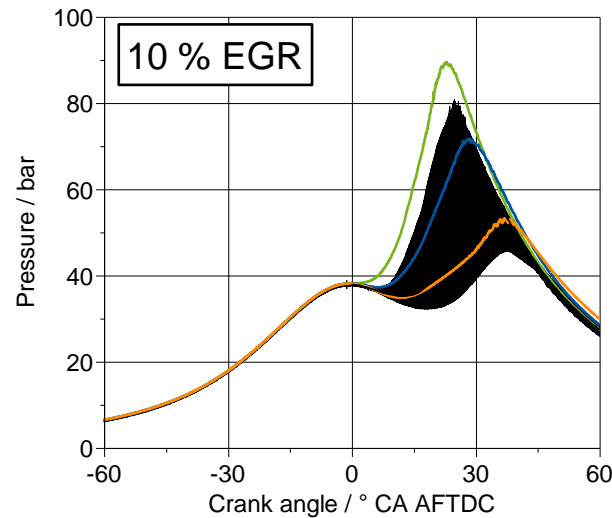
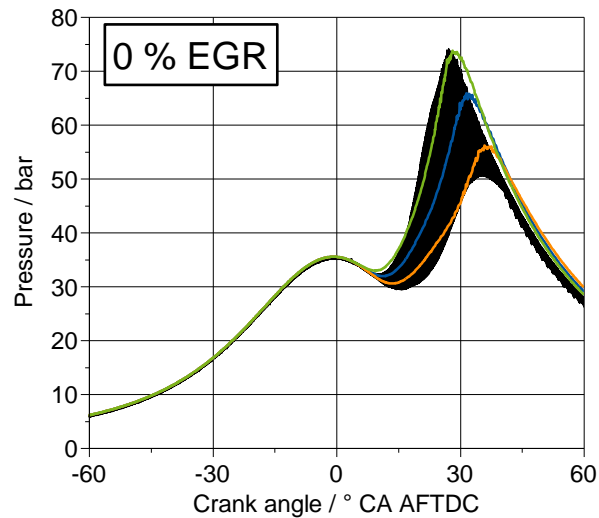


- ⊙ At high temperatures hydrogen peroxide reacts to two OH-radicals which significantly increase overall reaction velocity
- ⊙ In this process the high temperature ignition reactions are triggered and  $\text{CH}_2\text{O}$  is consumed

# Numerical investigation

## Good Match of Combustion and Knock with EGR at 1500 1/min as well

SIMULATION RESULTS OF 1500 1/MIN, 0-20 % EGR,  $T_{\text{INTAKE}} = 35 \text{ }^\circ\text{C}$



# Agenda/Content

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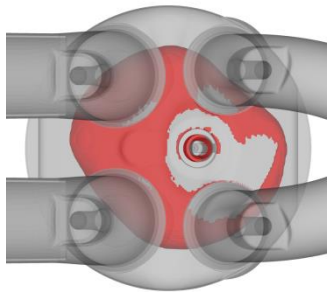
- Introduction
- Experimental investigation
- Numerical investigation
- Summary and Outlook

# Summary

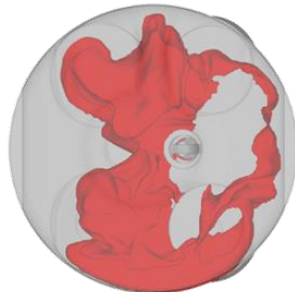
## Good performance of RANS simulation methodology

### KEY FINDINGS OF NUMERICAL INVESTIGATION

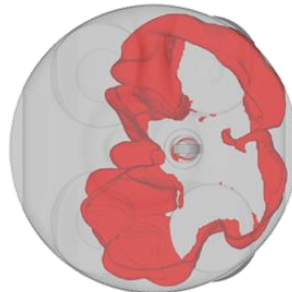
- ④ A RANS simulation model with coupled reaction mechanism and a validated gasoline surrogate was build up
- ④ Simulation methodology was able to successfully calculate the combustion under the influence of EGR and changing turbulence
- ④ Although cycle-to-cycle variation cannot be resolved with a RANS approach the knock limit could well reproduced with spark timing sweeps in the simulation model at most operating points
- ④ First stage ignition is playing an important role in the unburned zone and is affecting the temperature history of the unburned gas
- ④ This is especially true for combustion with EGR since the time of first stage ignition is only slightly affected by EGR
- ④ Possible improvement of methodology with large eddy simulation?



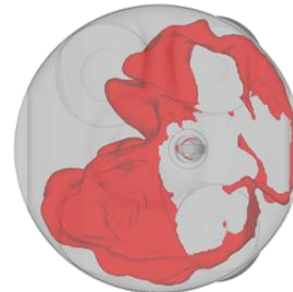
RANS



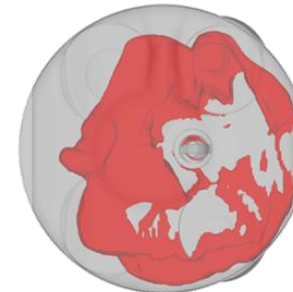
Cycle #1



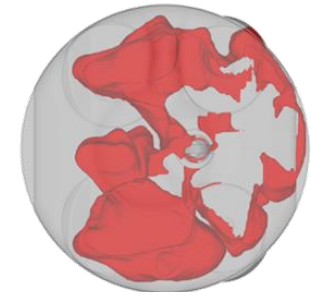
Cycle #2



Cycle #3



Cycle #4

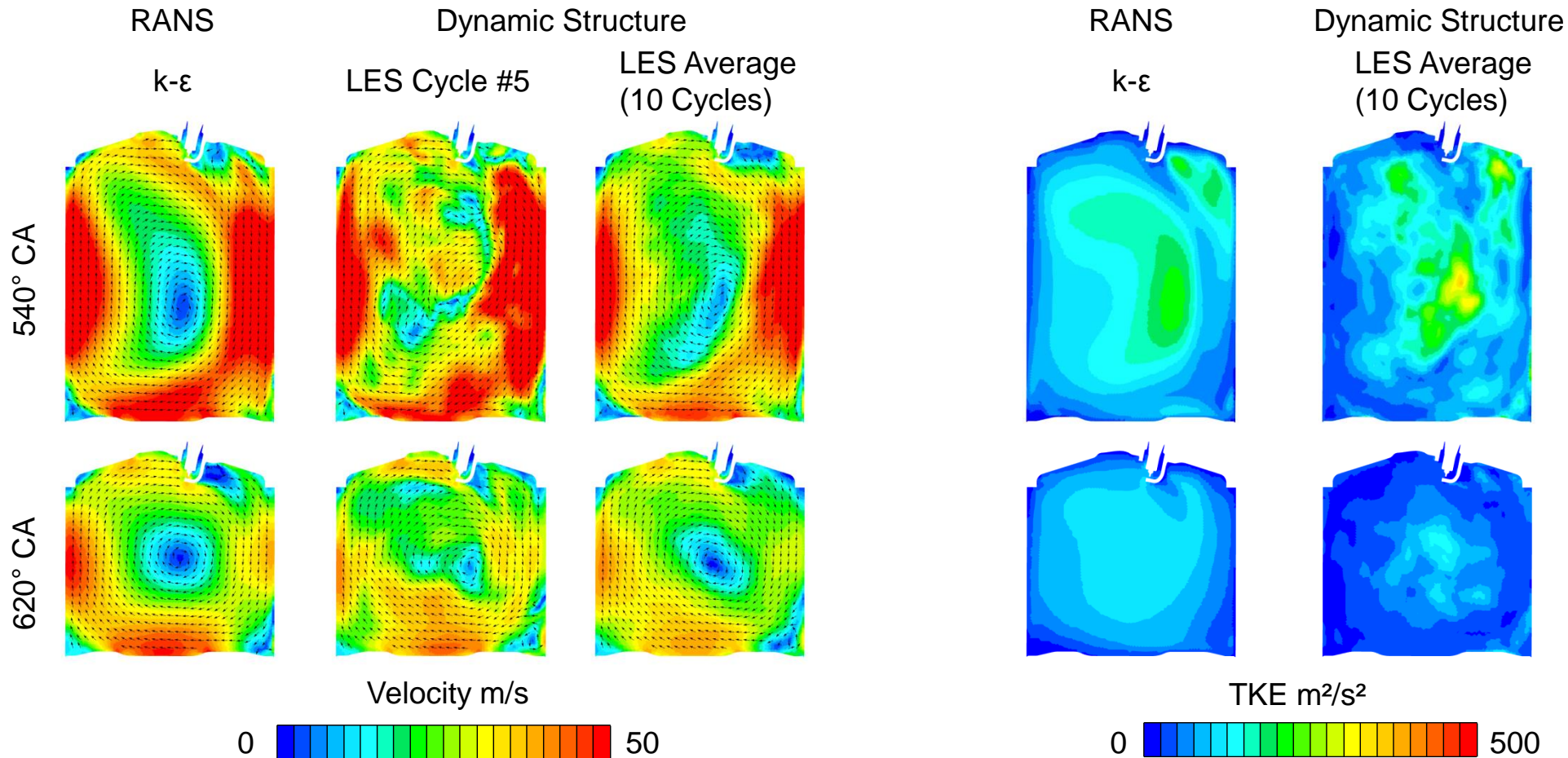


Cycle #5

# Outlook

## Large eddy simulation of ten individual cycles with dynamic structure model

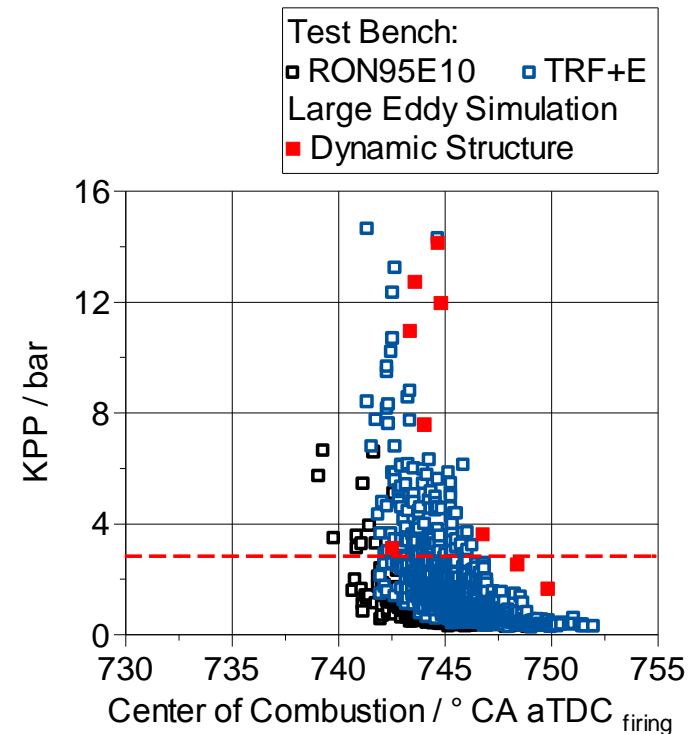
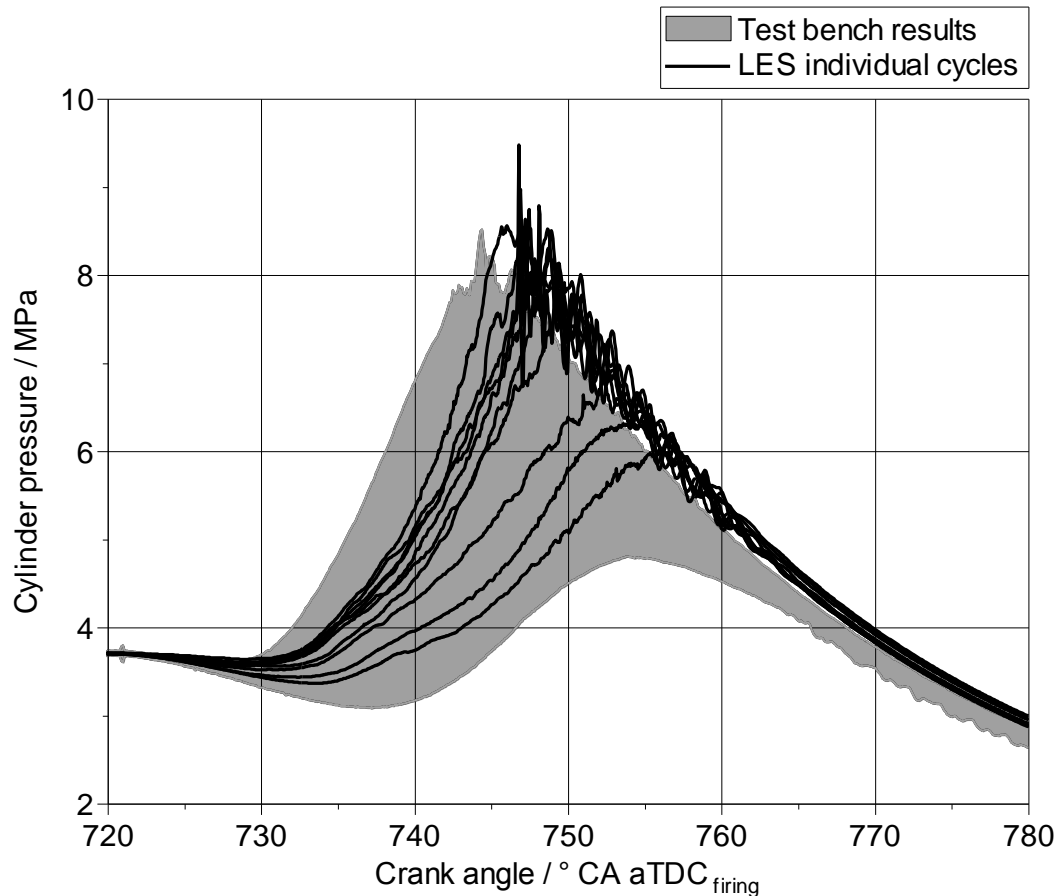
SIMULATION RESULTS OF 2500 1/MIN AND 0 % EGR: FAIR MATCH OF FLOW FIELD AND TKE ON AVERAGE



# Outlook

## Good match of fluctuation in combustion

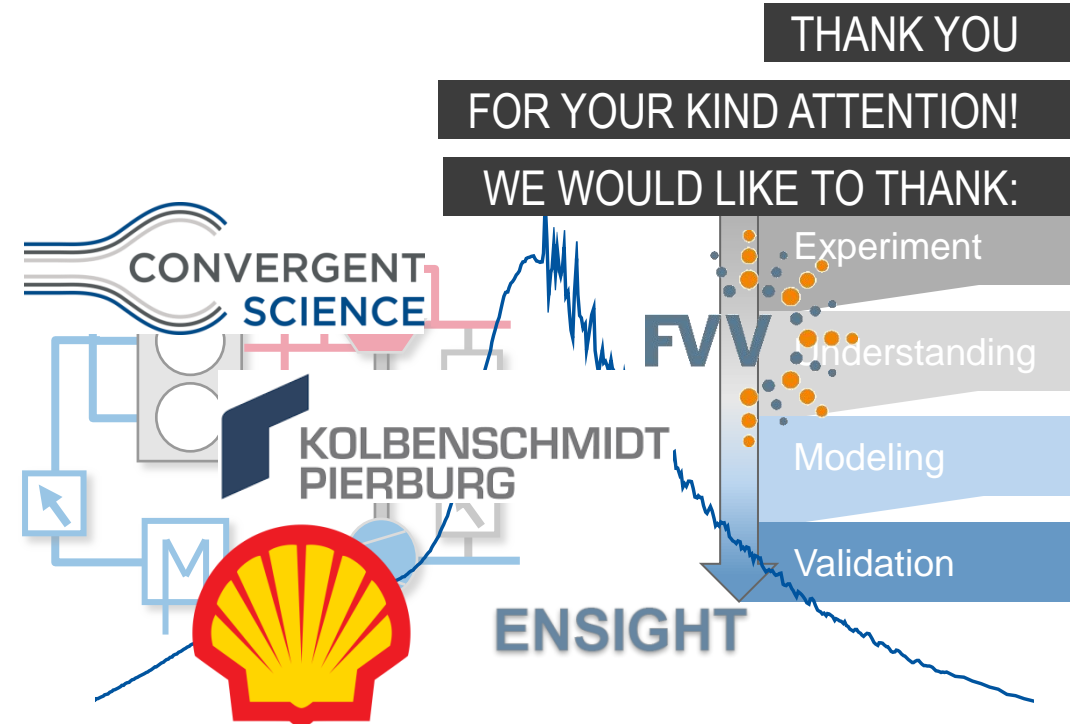
HIGHER KNOCK TENDENCY OF SURROGATE AT THIS POINT IS REFLECTED IN THE SIMULATION MODEL



LES yields promising results for future investigations

Special thanks go to FVV for the financial funding as well as to the working group members under the direction of Dr.-Ing. A. Kulzer (Porsche).

Furthermore we give special thanks to Convergent Science Inc., Shell, Kolbenschmidt-Pierburg and EnSight



Bologna, March 20<sup>th</sup>, 2018

Max Mally

Dr.-Ing. Marco Günther

Prof. Dr.-Ing. Stefan Pischinger